STEAM CONDENSATE RECOVERY OPPORTUNITY:

Museum Main Building steam condensate is being discharged to sewer. The plumbing code requires that the condensate be cooled before being discharged to the sewer. This cooling is being done by mixing potable water with the condensate. This cost saving opportunity is to use the water being sent to drain for makeup water in the cooling towers. This opportunity would result in water, and sewer cost savings.

	<i>\$</i> := ¤	
Given Data:	$WaterCost\$:= \frac{3.81 \cdot \$}{}$	
Water Rate \$3.81/1000 gallons	1000 <i>gal</i>	
Sewer Rate \$4.34/1000 gallons		
Steam/Energy Rate \$20.54/1000 pounds	SewerCost $:=$ $\frac{4.34 \cdot \$}{}$	
Electric Rate \$0.0553/KWH	1000 · gal	
Annual Operating Hours (AOH) 8760		
Annual Steam Consumed (ASC)	$E_{uclCost^{\&}} = 20.54$ \$	
Average Steam Flow 9,178 pounds/hour	$10^{6} \cdot Btu$	
	AOII 9760 hm	
	$AOH \coloneqq 8700 \cdot hr$	
Accumulioner	Electric Gente 0.0553 \$	
Assumptions:	$Electric Ost \mathfrak{s} := \frac{1}{kW \cdot hr}$	
1. Condensate is equal to 67% of steam use in		
2016-2017 due to steam losses from steam vents	la lb love	
and humidification steam and proposed	$ASC := 9178 \cdot \frac{1}{hr} \cdot AOH$	
reductions in the number of steam heating coils.		
2. Condensate water recovery can be used in	$ASC = (8.04 \cdot 10^{\circ}) \ b$	
cooling towers.	$Condensate := 0.67 \cdot \underline{\qquad \qquad }$	
3. Condensate Temperature 180 F.	8.34 •	
4. Potable Water Temperature 55 F.	gal	
5. Condensate is cooled to 110 F for sewer		
discharge.	Condensate = 6458935 gal	
6. Cooling tower condenser water flow rate, 2800		
GPM (TFR).	$TFR \coloneqq 2800 \cdot \frac{gat}{c}$	
7. Cooling tower cycles of concentration, 3 (C).	min	
8. Cooling tower drift rate, 0.005%, (D)		
9. Cooling tower operating power, 0.4 kW/ton, (TP)	C := 3 $D := 0.00005$	
10. Evaporation Rate 1% of flow rate.		
11. Cooling tower condenser water supply	$TP := 0.4 \cdot \frac{kW}{kW}$	
temperature to chiller, 75 F.	12000, Btu	
	hr	

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Cooling Tower Water Use (TWU):					
$Evaporation \coloneqq 0.01 \cdot TFR$	$Evaporation = 28 \frac{gal}{min}$				
$Blowdown \coloneqq \frac{Evaporation}{(C-1)}$	$Blowdown = 14 \frac{gal}{min}$				
(C-1)					
$Drift \coloneqq D \cdot TFR$	$Drift = 0.14 \frac{gal}{min}$				
$TWU \!\coloneqq\! Evaporation + Blowdown + Drift$	$TWU = 42.14 \underline{gal}$				
Condensate Cooling Water/Sewer (CWS):					
Cooling water and sewer required to cool condensate from 180 F to 110 F using potable water at 55 F. Required by plumbing code before condensate can be sent to drain system.					
$CWS \coloneqq \frac{Condensate}{AOH} \left(\frac{180 - 110}{110 - 55} \right)$	$CWS = 15.64 \frac{gal}{min} CWS \cdot AOH$				
	$CWS \cdot AOH = \left(8.22 \cdot 10^6\right) $ gal				

Available Cooling Water plus Condensate (AW):

$AW := CWS + \frac{Condensate}{CONDENSATE}$	AW = 27.929 gal
AOH	min

Annual Water/Sewer Cost Savings:

 $WaterSewerCost\$ \coloneqq WaterCost\$ + SewerCost\$$

Savings := $CWS \cdot AOH \cdot (WaterSewerCost$) + $Condensate \cdot (WaterSewerCost$)

 $Savings = (1.196 \cdot 10^5)$ \$119,637.09

Electric cost to run water pumps to pump condensate to tower \$ (EC) :

Pump break horsepower, GPM=pump gallons per minute, TDH=pump total dynamic head feet water, SG=specific gravity of water 1 and E= pump efficiency 50 percent

$TDH \coloneqq 115$	$SG \coloneqq 1$ $E \coloneqq 0.5$		
$gpm_{avg} \coloneqq 27.93$	$BHP_{pump} \coloneqq \frac{gpm_{av}}{3}$	$\frac{g \cdot TDH \cdot SG}{960 \cdot E}$	$BHP_{pump} = 1.622$
$EC_{PC} \coloneqq \left\langle BHP_{pur} \right\rangle$	$_{np} \big) \cdot 0.746 \ \mathbf{kW} \ AOH \cdot Electr$	icCost\$	$EC_{PC} = 586.235 \$
$PumpElectric \coloneqq$	$BHP_{pump} \cdot 0.746 \cdot kW$	PumpElectric = 1.	21 kW

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